

**Introduction to Finite Volume Methods – II**  
**Prof. Ashoke De**  
**Department of Aerospace Engineering**  
**Indian Institute of Technology, Kanpur**

**Lecture – 36**  
**Fluid Flow Computation: Incompressible Flows – VI**

So, welcome back to the lecture series of Finite Volume.

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**Fluid Flow problems: incompressible**


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$$D_f = \frac{(D_f^u s_f^u)^u + (D_f^v s_f^v)^v + (D_f^w s_f^w)^w}{d_{cf}^u D_f^u s_f^u + d_{cf}^v D_f^v s_f^v + d_{cf}^w D_f^w s_f^w}$$


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Collocated SIMPE algorithm

1. Start with guess value:  $p^{(n)}, u^{(n)}, m^{(n)}$
2. Mom:  $\rightarrow v^*$
3. Update  $\rightarrow m$  using Rhie-Chow interpolation.
4. Solve  $\rightarrow p'$
5. Update  $\rightarrow u, p', m$
6. Treat the obtained soln to be new guess.
7. Move to next time level integration

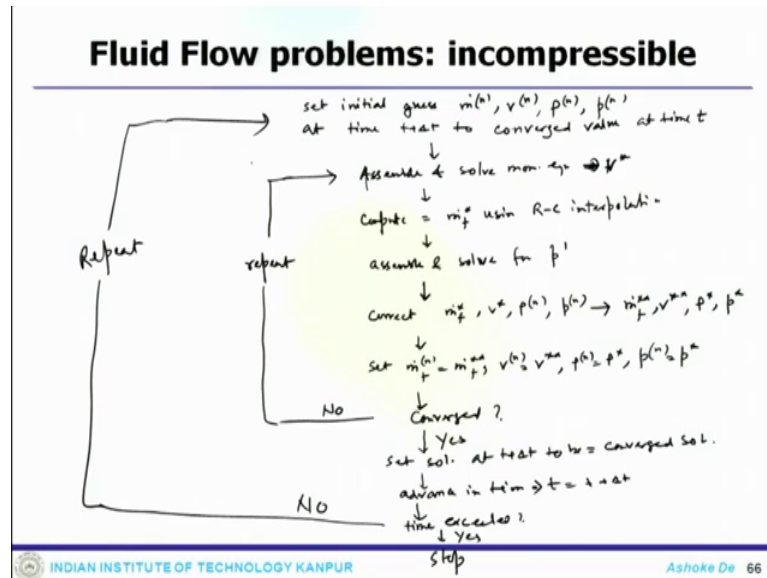

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And, where we will continue our discussion where we left in the last lecture. So, if you put the collocated algorithm together. So, collocated simple algorithm so, what it says that so, again you start with guess values which is  $p_n, u_n, m \cdot n$ . Then, solve the momentum equation to get  $V^*$  then update the  $m \cdot n$  using Rhie-Chow interpolation and compute the momentum satisfying mass field. Fourth; now, you solve the pressure correction equation to get  $p'$ .

Now, with the pressure correction field so now, you update velocity and pressure field,  $u$  and pressure field and also the mass flow rate which is  $u \cdot \text{double star } p \cdot \text{star } m \cdot \text{dot double star}$  and check. So, now you can treat this new solution to be a the updated solution to be the new guess. If it is mass conserving so, treat the obtained solution to be new guess. If it satisfy everything stop here otherwise you can go back to step here and repeated once you get a converge solution then move for move to next time level integration.

So, in every time level you carry out this process to get a converge solution and then you keep doing this. So, you can someone put the flow chart which will look quite nice.

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Let say set initial guess which is  $m \cdot n, v_n, \rho_n, p_n$  at time  $t$  plus  $\Delta t$  to converged value converged value at time  $t$ . Then, from there you assemble and solve momentum equation for to get  $V^*$  from their one can compute  $m \cdot f$  using Rhie-Chow interpolation  $m \cdot f^*$ . Then, assemble and solve for  $p$  point pressure correction from their you can correct  $m \cdot f^* v^* \rho_n$  and  $p_n$  to get  $m \cdot f^* v^* \rho^* p^*$ .

Now, you can set  $m \cdot f^* v^* \rho_n$  equals to  $m \cdot f^* v^* \rho^* p^*$ . So, converged if no, then from here you go back to this process and repeat. So, if this is no, if it is not converged then go back and repeat this process. Now, if it is yes, then set solution at  $t$  plus  $\Delta t$  to be equal to the converged solution now you advance in time and set  $t$  equals to  $t$  plus  $\Delta t$ .

Now, if time exceeded, yes, to stop it will get you stop if no from here you go back and repeat. So, this is no. So, this goes in physical iteration and within that this is a iterative process and this is how you get solution for the simple algorithm get working, so, for this collocated arrangement.

Now, moving ahead once you get a system ready then it is important to also see how one can implement the boundary condition.

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### Fluid Flow problems: incompressible

Rhie-Chow interpolation  $\rightarrow$  modified at boundary

$$\bar{v}_b = \bar{v}_c$$

$$\bar{v}_b^* = v_c^*$$

$$\nabla p_b^{(n)} = \nabla p_c^{(n)}$$

$$D_b^v = D_c^v$$

$$v_b^* = \bar{v}_b - D_c^v \left( \nabla p_b^{(n)} - \nabla p_c^{(n)} \right)$$

$$= v_c^* - D_c^v \left( \nabla p_b^{(n)} - \nabla p_c^{(n)} \right)$$

R-C interpolation

$$m_b = \rho_b v_b^* \cdot s_b$$

$$= \rho_b \left[ v_c^* - D_c^v \left( \nabla p_b^{(n)} - \nabla p_c^{(n)} \right) \right] \cdot s_b$$

$$= \rho_b v_c^* \cdot s_b - \rho_b D_c^v \left( \nabla p_b^{(n)} \cdot s_b - \nabla p_c^{(n)} \cdot s_b \right)$$

example of boundary element

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So, to do that let us start working on the boundary condition and this is an example of an boundary element. So, this is an example of an boundary element where you can see this is the boundary face and this is the cell center and this is the direction of that and this is the surface vector.

Now, one important thing is that when the face located at a boundary this is the boundary face has to provide the boundary condition now the first thing to be of interest is the expression of the Rhie-Chow interpolation at the boundary face. So, which now the Rhie-Chow interpolation at boundary face needs to be modified it needs to be modified at boundary.

So, what one can write that any variable which is at boundary is variable of the c. So, b refers to the boundary face and in that way the adopting this practice error is the Rhie-Chow interpolation so, at the boundary values can be written as  $b_c \star \Delta p_b$  at n equals to  $\Delta p_c$  at n and  $D_b^v$  equals to  $D_c^v$ . So, this one can do and your  $v_b^*$  equals to  $v_b^* - D_c^v \Delta p_b$  at n minus  $\Delta p_b$  at n and this is your standard Rhie-Chow interpolation which one can write minus  $D_c^v \Delta p_b$  at n minus  $\Delta p_b$  at n which is at the boundary face Rhie-Chow interpolation.

And, the mass flux which can be written as  $m \cdot b$  equals to  $\rho \cdot V \cdot S \cdot b$  which is  $\rho \cdot V \cdot c \cdot \Delta t \cdot \Delta p \cdot b \cdot n \cdot \Delta t \cdot S \cdot b$ . So, if you expand this would be  $V \cdot \Delta t \cdot S \cdot b \cdot \Delta p \cdot n \cdot \Delta t \cdot S \cdot b$  minus  $\rho \cdot V \cdot D \cdot c \cdot \Delta t \cdot \Delta p \cdot b \cdot n \cdot \Delta t \cdot S \cdot b$ . So, this is a component which comes along this line and the tangential of the default correction along the normal line. So, that has this component. So, this is an implementation of the boundary condition which is presented for the momentum equation.

Now, secondly, we can look at the boundary condition implementation for the pressure equation or the pressure correction equation and for the cases when the boundary condition for the momentum and pressure corrections equation are codependent then you need the full treatment of the pressure correction equation.

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**Fluid Flow problems: incompressible**

Mom. eqn: Semi-discretized form:

$$\frac{(\rho V_c - \rho V_c^0)}{\Delta t} V_c + \sum_{f \in \text{nb}(c)} (m_f V_f) = - \sum_{f \in \text{nb}(c)} (p_f S_f) + \sum_{f \in \text{nb}(c)} (\tau_f \cdot S_f) + B$$

$$\sum_{f \in \text{nb}(c)} m_f V_f = \sum_{f \in \text{interior nb}(c)} m_f V_f + \underbrace{m_b V_b}_{\text{boundary face}}$$

$$\sum_{f \in \text{nb}(c)} (\tau_f \cdot S_f) = \sum_{f \in \text{interior nb}(c)} (\tau_f \cdot S_f) + \underbrace{\tau_b \cdot S_b}_{\text{at boundary face}}$$

$$= \sum_{f \in \text{int. nb}(c)} (\tau_f \cdot S_f) + F_b$$

$$\sum_{f \in \text{nb}(c)} (p_f S_f) = \sum_{f \in \text{int. nb}(c)} (p_f S_f) + \underbrace{p_b S_b}$$

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Now, momentum equations once we look at it. So, for momentum equation the semi-discretized semi-discretize form looks like  $\rho \cdot V \cdot c \cdot \Delta t \cdot \Delta p \cdot b \cdot n \cdot \Delta t \cdot S \cdot b$  which is previous time step, so, plus summation  $f \in \text{nb}(c) \cdot m \cdot f \cdot V \cdot f$ .

So, this is elemental discretisation face integration equals to summation of  $f \in \text{nb}(c) \cdot p \cdot f \cdot S \cdot f$  this is again face discretisation plus summation of  $\tau \cdot f \cdot \dot{S} \cdot f$  this is also face plus B, which is again elemental discretisation. Now, these are already taken care of individually. So, straight away what we can write for this face which is  $m \cdot \dot{f} \cdot V \cdot f$  one can write that interior  $f \in \text{nb}(c) \cdot m \cdot \dot{f} \cdot V \cdot f$  plus  $m \cdot \dot{b} \cdot V \cdot b$  which is at boundary face and these are all

interior face. So, you can decompose into two component and similarly one can decompose the component of the stress component tensor which is again  $f$  at the interior  $n$   $b$   $c$   $\tau$   $f$  dot  $S$   $f$  plus  $\tau$   $b$  dot  $S$   $b$ .

So, this is at boundary face which one can write summation of  $f$  interior  $n$   $b$   $c$   $\tau$   $f$  dot  $S$   $f$  plus as a source term and similarly the pressure discretisation of the boundary face one can write  $f$  interior faces  $p$   $f$   $S$   $f$  plus  $p$   $b$   $S$   $b$ . So, this is at boundary face. So, one has to look at individual component and then treat the boundary values.

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**Fluid Flow problems: incompressible**

Wall - B.C (No slip)

$\rho_b = ?$   $m_b = 0$   
 $v_b = v_{wall}$

$F_b = F_{\perp} + F_{\parallel}$   
 normal dir.  $\rightarrow$  tangential dir. to wall

$F_b = F_{\parallel} = \tau_{wall} S_b$ ,  $\tau_{wall} = -\mu \frac{\partial v_{\parallel}}{\partial d_{\perp}}$   
 $\frac{\partial v_{\parallel}}{\partial d_{\perp}} \rightarrow$  parallel to wall  
 $\rightarrow$  normal dir.

$n = \frac{S_b}{S_b} = (n_x, n_y, n_z)$ ,  $d_{\perp} = d_{cb} \cdot n = \frac{d_{cb} \cdot S_b}{S_b}$

$v_{\parallel} = v - (v \cdot n)n = \begin{cases} u - (un_x + vn_y + wn_z)n_x \\ v - (un_x + vn_y + wn_z)n_y \\ w - (un_x + vn_y + wn_z)n_z \end{cases}$

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So, for example, now if you come to wall boundary condition; now this is wall boundary condition what you have? You have no slip wall. So, you can see this is my face or the boundary face and the no slip wall condition means what you do not know  $p$   $b$ , but you know mass flux is 0 and you know the velocity of the boundary face is wall. So, this is the boundary face.

Now, a no slip boundary condition means you have a velocity, but typically this would be 0. So, now, what one can do this the  $F$   $b$  that can be expanded as a plus a parallel component. So, the  $F$   $b$  can be extended on the fluid can be written as this where this guy represents the tangential direction to the wall tangential direction to wall and this is the normal direction. So, which means this would be along this direction normal and this is along the tangential direction.

So, the  $F_b$  would be  $F$  this. So, in the normal direction which is supposed to be actually 0. So, the parallel direction would be  $\tau_{wall}$ . Now,  $\tau_{wall}$  is the shear state all wall which can be calculated as  $\mu \frac{dV}{dx}$  tangential to the directions by  $\frac{d}{dx}$  perpendicular direction. So,  $V$  is the velocity vector parallel to the wall. So, this is a parallel to the wall and  $d$  perpendicular is the normal distance.

So, this is how so, this is the parallel to wall and this is the normal distance. So, unit vector would be  $\frac{S_b}{|S_b|}$  where  $n_x, n_y, n_z$  would be the component then  $d$  perpendicular would be  $d \cdot \frac{S_b}{|S_b|}$  which is  $d \cdot \frac{S_b}{|S_b|}$  and  $V$  parallel would be  $V - (V \cdot n) \frac{n}{|n|}$  which will let you have  $u - u n_x + v n_y + w n_z$  and  $n_x v - u n_x, v n_y, w n_z$  and  $w - u n_x, v n_y, w n_z$ . So, this is how you compute all this component.

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### Fluid Flow problems: incompressible

$$\tau_{wall} = -\mu_b \frac{(v_c - v_b)_{||}}{d_{\perp}} = -\mu_b \frac{(v_c - v_b) - [(v_c - v_b) \cdot n] \frac{n}{|n|}}{d_{\perp}}$$


$$= -\frac{\mu_b}{d_{\perp}} \left\{ \begin{array}{l} (u_c - u_b) - [(u_c - u_b)n_x + (v_c - v_b)n_y + (w_c - w_b)n_z] n_x \\ (v_c - v_b) - [ \quad \quad \quad ] n_y \\ (w_c - w_b) - [ \quad \quad \quad ] n_z \end{array} \right\}$$

$n = \text{const.}$

$$a_c^u \leftarrow a_c^u + \frac{\mu_b S_b}{d_{\perp}} (1 - n_x^2) \quad \text{Continuity due to boundary face}$$

$$0 \leftarrow a_{F=b}$$

$$b_c^u \leftarrow b_c^u + \frac{\mu_b S_b}{d_{\perp}} [u_b(1 - n_x^2) + (v_c - v_b)n_y n_z - (v_c - w_b)n_z n_y] - \frac{1}{2} \mu_b S_b^3$$


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And, you can calculate the  $\tau_{wall}$  which would be  $\mu_b (v_c - v_b)_{||}$  which is parallel  $d$  perpendicular  $\mu_b$  which can be written as  $(v_c - v_b)_{||} = (v_c - v_b) - (v_c - v_b) \cdot n \frac{n}{|n|}$  and this is  $d$  perpendicular and you get all this component like  $u_c - u_b - u_c n_x + u_b n_x + v_c n_y - v_b n_y + w_c n_z - w_b n_z$  multiplied with  $n_x$ . So, this term would be common.

So, one can write similarly  $(v_c - v_b)_{||}$  this is the same term multiplied with  $n_y$  and  $(w_c - w_b)_{||}$  same term multiplied with  $n_z$ . So, that is what one get.

Now, for laminar flow; now this is what one can obtain. Now, another the thing is that u component direction the coefficients of the boundary elements for the momentum equation. For u-component in that direction your a c u would be written as a c u plus mu b S b by d perpendicular one minus n x square. So, this is the contribution due to boundary face and this is interior face similarly a F u equals to b 0 and b c u would be b c u plus mu b by d perpendicular S b u b 1 minus n x square plus V c minus V b n y n x minus w c minus w b n z n x minus p b S b x. So, this is the contribution come from the boundary and this is from the interior face.

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**Fluid Flow problems: incompressible**

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v - comp., w - comp.

$$p_b \rightarrow p_b = p_c + \nabla p_c^{(n)} \cdot d_{cb}$$

$$\dot{m}_b^* = \rho_b v_b^* \cdot S_b - \rho_b D_c^v (\nabla p_b^{(n)} - \nabla p_c^{(n)}) \cdot S_b$$

at wall  $\rightarrow \dot{m} = 0, v = 0$

$$0 = 0 - \rho_b D_c^v (\nabla p_b^{(n)} - \nabla p_c^{(n)}) \cdot S_b$$

$$D_c^v \nabla p_b^{(n)} \cdot S_b = \nabla p_b^{(n)} \cdot S_b' = \nabla p_c^{(n)} \cdot S_b'$$

$$S_b' = E_b + T_b, \quad \nabla p_b^{(n)} \cdot (E_b + T_b) = \nabla p_c^{(n)} \cdot S_b'$$

$$\nabla D_c (p_b - p_c) = \nabla p_c^{(n)} \cdot S_b' - \nabla p_b^{(n)} \cdot T_b$$

$$p_b = p_c + \frac{\nabla p_c^{(n)} \cdot S_b' - \nabla p_b^{(n)} \cdot T_b}{D_c}$$

Similarly, one can actually get the V component and w component. So, it can be obtained similarly. All the matrixes are already been written. Now, the important point which comes with that at the boundary the pressure p b which is unknown and it needs to be extrapolated from the interior solution. So, one can use some sort of a Taylor series expansion and write that p b equals to P c plus delta p c dot d c b. Now, the mass flow rate which is expressed as Rhie-Chow interpolation which will have the contribution like this delta p b n minus delta p c n dot S b.

Now, the mass flow rate and the velocity at wall boundary at wall the mass flow rate is 0 velocity is also 0 then this guy walls down to 0 equals to 0 minus rho b D c v minus delta p b n minus delta p c n dot S b. So, which one can modify it and like right that n dot S b equals to delta p b n dot like this which is dealt p c n dot S b. Now, S b has two

component like  $E_b + T_b$ . So, one can write the term  $\Delta p_b n \cdot E_b + T_b$  which will get you the  $\Delta p_c n S_b$ . So, one can write  $D_c p_b - p_c$  equals to  $\Delta p_c n \cdot S_b$  minus  $\Delta p_b n \cdot T_b$ . So, all these one you put together you get  $p_b$  equals to  $p_c$  plus  $\Delta p_c$  at previous time iteration dot minus  $p_b n \cdot t_b$  by  $D_c$ .

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**Fluid Flow problems: incompressible**

slip wall -  $p_b = ?$   
 $m_b = 0$   
 $F_b = 0$

$a_c^v \leftarrow a_c^v$   
Interior face

$0 \leftarrow a_{f=b}^v$

$b_c^v \leftarrow b_c^v - p_b S_b$   
Interior face      boundary face

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Now, similarly one can have slip wall. So, this is an example of slip wall and at slip wall what happens your  $p_b$  is still unknown, mass flow rate is 0, force is 0. So, what one can modify that coefficients would be like this  $a_c$ . So, this is only contribution come from interior faces and  $a_f$  equals to  $b$  which 0 and  $b_c$  is  $b$  minus  $b$  which is again interior faces and boundary faces.

So, one can see once you write everything mathematically in this fashion then when you come to an particular application of the boundary condition then it becomes quite easier.



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### Fluid Flow problems: incompressible

Known vel.  $p_b = ?$ ,  $\dot{m}_b = \text{specified}$   
 $v_b = \text{specified}$

$$p_b = p_c + \nabla p_c \cdot \Delta c_b$$

$$a_c^v \leftarrow a_c^v$$

$$b_c^v \leftarrow b_c^v - a_{F=b}^v v_b$$

$$0 \leftarrow a_{F=b}^v$$

$$\left. \begin{array}{l} F_b = \tau_b \cdot S_b \\ \dot{m}_b v_b \end{array} \right\} \text{Calculate}$$

$v_b = v_{\text{specified}}$

$n$

$S_b$

$\rho_b = \rho, \nu_b, S_b$

$p_b = ?$

$v_c = v_{\text{specified}}$

specified vel.

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Now, this is a case where you have a specified velocity. So, when you have a known velocity which means  $p_b$  still unknown  $\dot{m}_b$  can be specified and  $v_b$  is specified. So, one that is the case then you can have this calculation like  $p_b$  equals to  $p_c$  plus delta  $p_c \cdot n \cdot \Delta c_b$  and your other things will get modified  $a_c^v$  is  $a_c^v$  and  $b_c^v$  which will be modified like  $a_{F=b}^v$  equals to  $b_c^v - a_{F=b}^v v_b$  and  $a_{F=b}^v$  equals to 0 and the convection mass flow rate  $\dot{m}_b v_b$  and diffusion term of the boundary face can be calculated.

So, like this  $F_b$  equals to  $\tau_b \cdot S_b$  and  $\dot{m}_b v_b$  this can be calculated as the given condition is known. Now, similarly one can have specified pressure.

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### Fluid Flow problems: incompressible

Pressure specified

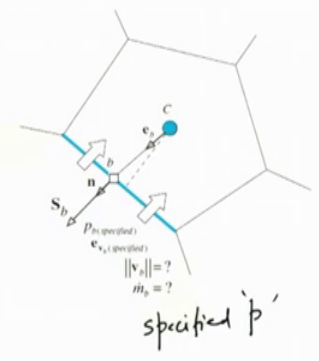
$$p_b = p_{\text{specified}}, \quad \dot{m}_b = ?$$

$$V_b = ?$$

$$\dot{m}_b^{\text{cor}} = \rho_b V_b^{\text{cor}} \cdot S_b$$

$$= \rho_b \parallel V_b^{\text{cor}} \parallel e_n \cdot S_b$$

$$\Rightarrow \parallel V_b^{\text{cor}} \parallel = \frac{\dot{m}_b^{\text{cor}}}{\rho_b (e_n \cdot S_b)}$$

$$= \parallel V_b^{\text{cor}} \parallel e_n$$


specified 'p'

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So, this case is a specified  $p$ . So, the pressure specified at the boundary which means my  $p_b$  is specified and  $\dot{m}_b$  is not known. So, velocity  $b$  is also not known. So, one has to modify the equation of the mass flow rate correction mass flow rate would be  $\rho_b b \cdot V_{\text{correction}} \cdot S_b$  which will be  $\rho_b V_b^{\text{star}} \cdot e_n \cdot S_b$ , which get you back the  $V_b^{\text{star}} = \dot{m}_b^{\text{star}} / (\rho_b e_n \cdot S_b)$  which is  $V_b^{\text{star}} e_n$ .

So, we can actually see the other kind of boundary conditions also in the next lecture. So, will stop here and take it up from here in the next lecture.

Thank you.