

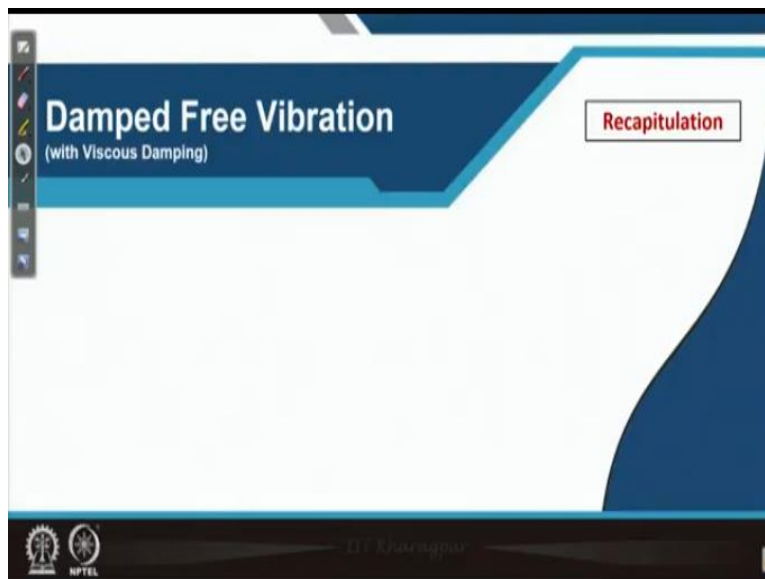
Soil Dynamics
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Lecture 4
Single Degree of Freedom System (SDOF) Part 3

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Hello friends, today, we will continue our last class.

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Last class, we have discussed about general solution of the damped system.

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Damped Free Vibration
(with Viscous Damping)

Recapitulation

➤ **General Solutions:**

- ✓ For Over-damped system: $z = A_1 e^{\beta_1 t} + A_2 e^{\beta_2 t}$
where, $\beta_1 = -D\omega_n + \omega_n\sqrt{D^2 - 1}$ and $\beta_2 = -D\omega_n - \omega_n\sqrt{D^2 - 1}$ for $D > 1$
- ✓ For Critically-damped system: $z = (A_1 + A_2 t)e^{-\omega_n t}$ for $D = 1$
- ✓ For under-damped system: $z = e^{-\omega_n D t} [A_1 \sin \omega_d t + A_2 \cos \omega_d t]$
where ω_n is undamped natural frequency and ω_d is damped natural frequency.
Also, $\omega_d = \omega_n \sqrt{1 - D^2}$ for $D < 1$

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where ω_n is undamped natural frequency and ω_d is damped natural frequency.
Also, $\omega_d = \omega_n \sqrt{1 - D^2}$ for $D < 1$

Handwritten notes:
 ω_n : undamped natural frequency
 D : Damping Ratio = $\frac{c}{c_c}$
 $c_c = 2\sqrt{mk}$
 ω_d : Damped natural frequency

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We have seen three types of damped system with viscous damping. First one was the over-damped system and the general solution for over-damped system is expressed by z is equal to A_1 times e to the power $\beta_1 t$ plus A_2 times e to the power $\beta_2 t$. What was β_1 and β_2 ? β_1 was we have seen β_1 was $-D\omega_n + \omega_n\sqrt{D^2 - 1}$. In this case D is greater than 1 that is the reason I have written here as square root of $D^2 - 1$.

At the second root was β_2 which is $-D \omega_n$, minus ω_n times square root of $D^2 - 1$. So, what is ω_n and D here? ω_n is natural or I can write it as undamped natural frequency, so ω_n is undamped natural frequency of the system, d is the damping ratio or damping factor which is expressed by the ratio of the coefficient of damping to the critical damping and critical damping we can calculate by $2 \times \text{square root of mass times (speed) stiffness}$. What is D (er) here? That is now clear.

Next to us critically damped system that we have discussed in the last class, so the general solution for critically damped system was found out Z is equal to $A_1 + A_2 t$ whole thing under bracket is multiplying by the $\omega_n e^{-\omega_n t}$. In this case since D is equal to 1 that is the reason here when writing $e^{-\omega_n t}$ I have not written D because D is equal to 1 here.

The third case was under-damped system and the general solution for under-damped system is Z which can be written by $e^{-\omega_n D t}$ times $A_1 \sin \omega_d t + A_2 \cos \omega_d t$. What is ω_d here? ω_d is damped natural frequency, and these damped natural frequency can be calculated by undamped natural frequency ω_n times square root of $1 - d^2$. In this case, d is less than 1 that is the reason I have written here $1 - d^2$ instead of writing $d^2 - 1$ in case of over-damped system.

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Damped Free Vibration
(Viscous Damping)

General solution for under-damped condition:

$$z = e^{-\omega_n D t} [A_1 \sin \omega_d t + A_2 \cos \omega_d t]$$
 where ω_n is undamped natural frequency and ω_d is damped natural frequency.

Fig. 5.1 Solution of under-damped system

Damped Free Vibration

(with Viscous Damping)

General solution for under-damped condition:

$$z = e^{-\omega_n D t} [A_1 \sin \omega_d t + A_2 \cos \omega_d t]$$

where ω_n is undamped natural frequency and ω_d is damped natural frequency.

$D < 1$
 $t_2 - t_1 = T = \frac{2\pi}{\omega_d}$

Fig. 5.1 Solution of under-damped system

So, now, today we will focus on the under-damped condition that means, damping ratio is less than 1 here you can see the general solution for under-damped system. What we can see over the time the amplitude of vibration reduces, let us take this amplitude that means the first peak in this diagram that is z_1 at time t equals to t_1 likewise the second peak which is your z_2 (that is nothing) that is occurred when t equals to t_2 .

Another interesting thing which we can note here is the difference between time t_2 and time t_1 that means t_2 minus t_1 is the time period for this type of system which is equal to 2π divided by ω_d , ω_d is nothing but the natural frequency of the damped system.

So, let us do some exercise now, what will be the expression for z_1 and z_2 from these that we will try to find out. So, now, we will see the expression for the z_1 and z_2 in this under-damped condition. So, let us go to the whiteboard. So, now, we will see (how we can write the magnitude?) how we can write the expression for z_1 and z_2 ? So, z_1 is the vibration at time t equals to t_1 and z_2 is the response at time t equals to t_2 .

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The whiteboard shows the following derivations:

$$z = e^{-\omega_n D t} [A_1 \sin \omega_d t + A_2 \cos \omega_d t]$$

at $t = t_1$

$$z = z_1 = e^{-\omega_n D t_1} [A_1 \sin \omega_d t_1 + A_2 \cos \omega_d t_1]$$

at $t = t_2$

$$z = z_2 = e^{-\omega_n D t_2} [A_1 \sin \omega_d t_2 + A_2 \cos \omega_d t_2]$$

$$z_2 = e^{-\omega_n D (t_1 + \frac{2\pi}{\omega_d})} [A_1 \sin \omega_d (t_1 + \frac{2\pi}{\omega_d}) + A_2 \cos (t_1 + \frac{2\pi}{\omega_d})]$$

$$= e^{-\omega_n D (t_1 + \frac{2\pi}{\omega_d})} [A_1 \sin (\omega_d t_1 + 2\pi) + A_2 \cos (\omega_d t_1 + 2\pi)]$$

$$= e^{-\omega_n D (t_1 + \frac{2\pi}{\omega_d})} [A_1 \sin \omega_d t_1 + A_2 \cos \omega_d t_1]$$

Definitions on the right side of the board:

$$\omega_d = \omega_n \sqrt{1 - D^2}$$

(Damped natural freq)

$$D = \text{Damping ratio} = \frac{c}{2\sqrt{mk}}$$

$$t_2 = t_1 + T = t_1 + \frac{2\pi}{\omega_d}$$

So, let us see we have the expression z is equal to e to the power minus $\omega_n D t$ times A_1 sine of $\omega_d t$ plus A_2 times cosine of $\omega_d t$. Now, here what is ω_d ? And what is D ? ω_d is ω_n times square root of $1 - D^2$ which is called as damped natural frequency likewise, D is called as damping ratio which is c divided by 2 times of capital MK , how it is coming that we have already seen in the last class.

Now, the Z_1 will be at t equals to t_1 , z is equal to Z_1 and that can be written as e to the power minus $\omega_n D t_1$ times A_1 sine of $\omega_d t_1$ plus A_2 times cosine of $\omega_d t_1$. Now, at time t equals to t_2 displacement Z is equal to Z_2 and that can be written as e to the power minus $\omega_n D t_2$ times A_1 sine $\omega_d t_2$ plus A_2 times cosine $\omega_d t_2$. Now, here we know the relationship between t_1 and t_2 what is it?

We know t_2 is equal to t_1 plus capital T , capital T is the time period. So, under under-damped condition capital T is 2π divided by ω_d . Therefore, Z_2 can be rewritten as e to the power minus $\omega_n D$ in place of t_2 . Now, we can write t_1 plus 2π divided by ω_d this thing will be multiplied by A_1 sine of ω_d in place of t_2 now we can write t_1 plus 2π by ω_d .

So, t_1 plus 2π by ω_d plus A_2 times cosine of t_1 plus 2π by ω_d , we can rewrite this expression now, as first term will remain as it is. Now, the second term within bracket can be

written as $A_1 \sin(\omega dt + 2\pi) + A_2 \cos(\omega dt + 2\pi)$ in the previous expression one ωd is missed, so please write it here. So, now, here we can write $A_2 \cos(\omega dt + 2\pi)$.

So, now, we know $\sin(2\pi + \theta)$ is nothing but $\sin \theta$ likewise $\cos(2\pi + \theta)$ is also $\cos \theta$. So, here then we can write $e^{-\omega dt + 2\pi}$ by ωd times $A_1 \sin(\omega dt + 2\pi) + A_2 \cos(\omega dt + 2\pi)$ please note here that the term within bracket is same as the term within bracket for Z_1 .

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The image shows a whiteboard with the following handwritten equations:

$$\frac{z_1}{z_2} = \frac{e^{-\omega dt}}{e^{-\omega dt + \frac{2\pi D}{\omega d}}}$$

$$= e^{2\pi D \left(\frac{\omega n}{\omega d}\right)} \quad \omega d = \omega n \sqrt{1-D^2}$$

$$\frac{z_1}{z_2} = e^{\frac{2\pi D}{\sqrt{1-D^2}}}$$

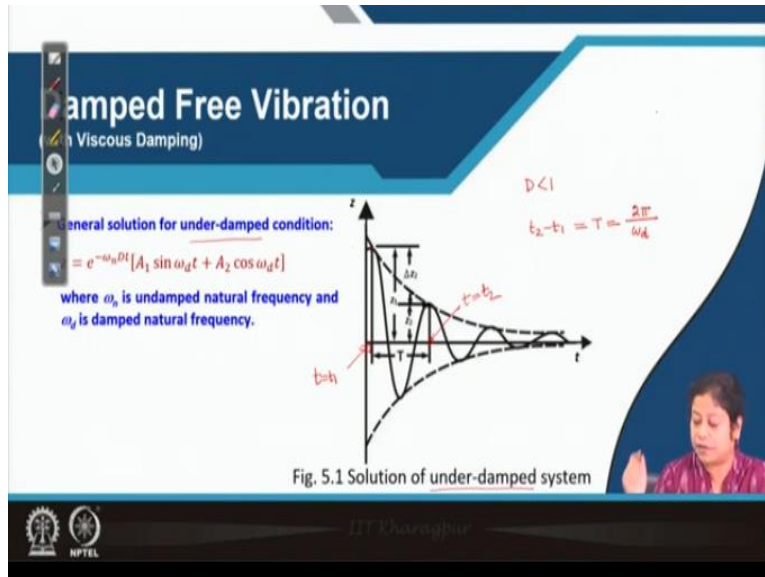
$$\ln\left(\frac{z_1}{z_2}\right) = \frac{2\pi D}{\sqrt{1-D^2}} = \delta = \text{logarithmic decrement}$$

Then we can write the ratio of Z_1 by Z_2 as $e^{-\omega nDt}$ since the term within bracket which is multiplied with this term for both numerator and denominator same, so I am not writing that term and that also is non-zero term. So, I can write it here as numerator already I have written and in the denominator, I can write $e^{-\omega nDt + 2\pi}$ by ωd finally, what we can write from this?

We can write from these $2\pi D \omega n$ times ωd . Now, we know ωd is equal to ωn sorry! Now, we know ωn or ωd actually ωd is equal to ωn times square root of $1 - D^2$. So, ωn by ωd will be one divided by square root of $1 - D^2$. So, we can write then Z_1 divided by Z_2 is equal to $2\pi D$ divided by square root of $1 - D^2$.

Now, if we will take log of Z1 divided by Z2, it will be 2 pi capital D divided by 1 minus D square and these sometimes called as logarithmic decrement it is called as logarithmic decrement.

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Now, another interesting thing we can see in this figure,, so here Z1 and Z2 are two consecutive amplitudes what will be happen, if instead of taking two consecutive peaks, we are taking second peak after n number of cycles, let us see.

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$$\frac{z_1}{z_2} = \frac{e^{-\omega_n D t_1}}{e^{-\omega_n D (t_1 + \frac{2\pi}{\omega_d})}}$$

$$= e^{2\pi D \left(\frac{\omega_n}{\omega_d}\right)}$$

$$\frac{z_1}{z_2} = e^{\frac{2\pi D}{1-D^2}}$$

$$\ln\left(\frac{z_1}{z_2}\right) = \frac{2\pi D}{1-D^2} = \delta = \text{logarithmic decrement}$$

$$z_{n+1} = e^{-\omega_n D t_{n+1}} [A_1 \sin \omega_d t_{n+1} + A_2 \cos \omega_d t_{n+1}]$$

$A_1 \sin \omega_d t_1 + A_2 \cos \omega_d t_1$
 $t_{n+1} = t_1 + nT$

So, in place of Z_2 , now, we will write the term Z_{n+1} because, we are now interested for the amplitude of peak after a number of cycles. So, what will be our equation? e to the power minus $\omega n D$ times t times at $n+1$ (it) cycle then rest of the thing within bracket can be written like $\sin \omega d t n + 1$ plus $A_2 \cos$ of $\omega d t n + 1$ and these term within bracket can be rewritten as...

I am just writing the term within bracket as again the same way $A_1 \sin \omega d t + 1$ because in this case the what is the relationship between t_1 and t_{n+1} ? In this case t_{n+1} is equal to $t_1 + n$ times of capital T , so we can write it this way only for the term within bracket.

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$$\frac{Z_1}{Z_{n+1}} = \frac{e^{-\omega n D t_1}}{e^{-\omega n D (t_1 + \frac{2\pi n}{\omega d})}}$$

$$= e^{-\omega n D (\frac{2\pi n}{\omega d})}$$

$$= e^{\left(\frac{2\pi n D}{\sqrt{1-D^2}}\right)}$$

$$\ln\left(\frac{Z_1}{Z_{n+1}}\right) = \frac{2\pi n D}{\sqrt{1-D^2}}$$

$$\delta = \frac{1}{n} \ln\left(\frac{Z_1}{Z_{n+1}}\right) = \frac{2\pi D}{\sqrt{1-D^2}}$$

Then what we can write for Z_1 divided by Z_{n+1} numerator it will be e to the power $\omega n d t + 1$ whereas, in the denominator we can write e to the power minus ωn times d times t_1 plus n times capital T or we can write it as $2\pi n D$ divided by ωd . So, finally, what we are getting is e to the power sorry! I am making mistake in writing.

So, finally, we can write it as e to the power minus $\omega n D$ and $2\pi n$ divided by ωd or we can write it also as e to the power minus $2\pi n$ capital D divided by square root of $1 - D^2$ square this is as power of e with the negative sign.

Then what will be log of Z_1 divided by Z_{n+1} ? It will be sorry! Here there will be no negative sign. So, the next step log of Z_1 divided by Z_{n+1} can be written as $2\pi n$ capital D

divided by 1 minus D square. So, in this case if we are interested to find out delta which is the logarithmic decrement then delta is equal to 1 upon n times log of Z1 divided by Zn plus 1 which is 2 pi D divided by 1 minus D square.

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Damped Free Vibration
(Viscous Damping)

General solution for under-damped condition:

$$z = e^{-\omega_n D t} [A_1 \sin \omega_d t + A_2 \cos \omega_d t]$$
 where ω_n is undamped natural frequency and ω_d is damped natural frequency.

▶ In Fig. 5.1, $\frac{z_1}{z_2} = e^{\frac{2\pi D}{\sqrt{1-D^2}}}$

▶ In other word, $\delta = \ln \left(\frac{z_1}{z_2} \right) = \frac{2\pi D}{\sqrt{1-D^2}}$

Fig. 5.1 Solution of under-damped system

So, now here we can see what I say that is written in short what is the logarithmic decrement delta? And what is the ratio of two consecutive peak, amplitudes of two consecutive peak?

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Numerical Problem-1

A vibration system consists of mass of 6 kg, a spring stiffness of 0.7 N/m and a dashpot with a damping coefficient of 2 N-s/m. Determine the critical damping and damping ratio.

Handwritten notes:
 $c_c = 2\sqrt{k \cdot m}$
 $D = \frac{c}{c_c}$

Handwritten calculations on a whiteboard:

$$m = 6 \text{ kg}$$

$$k = 0.7 \text{ N/m} \quad c = 2 \text{ N-s/m}$$

$$C_c \text{ (critical damping)} = 2\sqrt{mk} = 2\sqrt{(6)(0.7)} \text{ N-s/m (Ans)}$$

$$= 4.098 \text{ N-s/m}$$

$$D = \frac{c}{C_c} = \frac{2}{4.098} = 0.488 \text{ (Ans)}$$

Now, with this we can take one numerical problem to clear our understanding on under-damped condition and how to calculate logarithmic decrement damped natural frequency of a freely vibrating system with viscous damping etc. So, let us take this numerical problem. So, in this problem it is said that (vibration) a vibration system consists of mass of 6 kg sorry! Mass of 6 kg we can use the pen, a spring having stiffness 0.9 in Newton per meter and dashpot with a damping coefficient to Newton second per meter.

We are asked to determine the critical damping and damping ratio. So, critical damping means CC and damping ratio means capital D which is the ratio of C to CC. So, let us solve this problem, mass is 6 kg, K is this is K which is 0.7 Newton per meter and damping is 2 Newton second per meter, mass is 6 kg, then stiffness is 0.7 Newton per meter and damping coefficient of damping is how much 2 Newton second per meter.

We are asked to determine CC and capital D. So, critical damping means Sorry! Critical damping is CC not this one I am just rewriting the thing once again mass is 6 kg, then K is 0.7 Newton per meter and coefficient of damping is I think it was 2 Newton second per meter. So, CC which is the critical damping that is equal to 2 times of square root of capital in this case it is small m that means mass times stiffness.

So, mass is how much in this case? It is 6 kg unit is in SI system, K is 0.7 Newton per meter its unit is also in SI system so the unit for critical damping in this case will be in SI system which is

Newton second per meter. So, let us see what will be the final answer for this question? So, let me calculate it 2 times square root of 0.7 times 6, so it is coming something 4.098 Or we can take 9 also Newton second per meter this is critical damping.

Then damping ratio will be the ratio of the coefficient of damping to the critical damping. So, it will be 2 divided by 4.098 therefore, the final value of these is let me calculate 2 divided by 4.098 which is coming 0.488. So, the final answer for this problem is, this is answer one that critical damping and the second one is second answer which is for the damping ratio.

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Numerical Problem-1

A vibration system consists of mass of 6 kg, a spring stiffness of 0.7 N/m and a dashpot with a damping coefficient of 2 N-s/m. Determine the critical damping and damping ratio.

Ans. $c_c = 4.099$ N-s/m; $D = 0.488$

So, here I have written the final answer of these numerical problem, you can see CC is 4.099 actually the value which I got was 4.0987, so they have done the approximation at three decimal place. So, 4.099 Newton second per meter and capital D is 0.488.

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The slide is titled "Numerical Problem-2" and contains the following text: "A mass of 1 kg is to be supported on a spring having a stiffness of 980 N/m. The damping coefficient is 6.26 N-s/m. Determine the undamped natural frequency of the system. Also calculate the logarithmic decrement and the amplitude after three cycles if the initial displacement is 0.3 mm." Handwritten red annotations include: "K" above "stiffness", "C" above "damping coefficient", "ω_n" above "undamped natural frequency", "δ" above "logarithmic decrement", and "Z₄" above "amplitude after three cycles". The NPTEL logo is visible in the bottom left corner.

Now, let us see another numerical problem. So, in this numerical problem, it is said that a mass of 1 kg is to be supported on a spring having a stiffness 980 Newton per meter. The damping coefficient is also given which is 6.26 Newton second per meter. So, damping coefficient means it is C and stiffness means it is K. So, we are asked to determine the undamped natural frequency of the system.

So, we can calculate either ω_n or f_n or also we are asked to calculate the logarithmic decrement which is δ here and the amplitude up after three cycles if the initial displacement is 0.3 millimeter, so initial displacement I can write as Z_0 . So, we are asked to find out it for Z_3 I can write it this way or I can write it is as Z_4 I can write it as Z_1 and this one as Z_4 .

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Given: $k = 980 \text{ N/m}$
 $c = 6.26 \text{ N-s/m}$
 $m = 1 \text{ kg}$

Damping Ratio (D) = $\frac{c}{2\sqrt{mk}} = \frac{6.26}{2\sqrt{(1)(980)}} = 0.099$

Undamped Natural frequency $\omega_n = \sqrt{\frac{k}{m}} = \sqrt{\frac{980}{1}} \text{ rad/s} = 31.30 \text{ rad/s}$

$f_n = \frac{\omega_n}{2\pi} = 4.98 \text{ Hz}$

$\delta = \frac{1}{n} \ln\left(\frac{z_1}{z_{1+n}}\right) = \frac{2\pi D}{\sqrt{1-D^2}} = \frac{2\pi(0.099)}{\sqrt{1-(0.099)^2}} = 0.625$

For $n = 3$

$\frac{z_1}{z_4} = \frac{z_1}{z_4} = e^{\left[\frac{2\pi(3)(0.099)}{\sqrt{1-0.099^2}}\right]} = 6.523$

$\Rightarrow z_4 = \frac{0.3 \text{ mm}}{6.523} = 0.046 \text{ mm}$

So, let us solve this numerical problem then, first I am writing what is given to us, given data K is given which is 980 Newton per meter, also coefficient of damping which is c is given that is 6.26 in Newton second per meter, the mass of the system m is 1 kg. Now, first thing we need to calculate net undamped natural frequency. So, undamped natural frequency here is that is ω_n is equal to square root of K by m .

So, K is 980 divided by 1 that is in a radian per second. So, how much it is coming? Let me see, it is coming 31.30 in radian per second. So, we can calculate f_n also which is ω_n by 2π , ω_n means here circular frequency, so this divided by 2π that is coming 4.98 in hertz. The next thing which is asked is logarithmic decrement. So, logarithmic decrement is δ that can be written as 1 upon n times natural log of Z_1 divided by Z_1 plus n .

And we have already seen that, that is equal to $2\pi D$ divided by square root of 1 minus D square. Now, here D is unknown, so we need to first calculate D . So, the data which is given to us from that we can calculate damping ratio D that is equal to C divided by 2 times mK . So, C is (9) sorry! 6.26 divided by two times square root of mK means this quantity and it is a unique place number, so let me calculate, so it is coming 0.099 this is the damping ratio.

Then we can find out logarithmic decrement here $2\pi \cdot 0.099$ divided by $1 - 0.099^2$ which is coming 2π times 0.099 divided by square root of $1 - 0.099^2$, so we are getting 0.625 this is logarithmic decrement. Now, in the next step we need to calculate the Z_1 plus n , so in this case n is 3 . So, Z_1 divided by Z_1 plus 3 or you can write it as Z_1 divided by Z_4 is equal to e to the power $2\pi n$ is 3 here, so $2\pi n d$, d is 0.099 here divided by square root of $1 - 0.099^2$.

So, how much will be this? This is nothing but let me calculate first e to the power $2\pi n$ (divide) into 0.099 divided by square root of $1 - 0.099^2$, so we are getting 6.522 or 523 . Already Z_1 was given I think yeah Z_1 is 0.3 in millimeter, so Z_4 is Z_1 which is 0.3 in millimeter divided by 6.523 which is coming 0.3 divided by 6.523 it is coming just one minute, so 0.3 divided by this is coming.

Let me check once again whether I have calculated this part correctly or not, then it will be easy for me to proceed further. I think it is calculated correctly 6.523 I am just taking and it is coming 0.45 sorry! 0.045 . There is another term here which is coming in my case more than five so I am writing it approximately, I am writing it as 0 sorry! 0.046 in millimeter. So, in this way we can calculate first the undamped natural frequency then logarithmic decrement then the Z_4 .

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Numerical Problem-2

A mass of 1 kg is to be supported on a spring having a stiffness of 980 N/m . The damping coefficient is 6.26 N-s/m . Determine the undamped natural frequency of the system. Also calculate the logarithmic decrement and the amplitude after three cycles if the initial displacement is 0.3 mm .

Ans. $\omega_n = 31.30 \text{ rad/s}$; $f_n = 4.98 \text{ Hz}$; $D = 0.099$; $\delta = 0.625$; $z_4 = 0.0452 \text{ mm}$

Now let us see what is the final solution for this problem? So, final solution is given ω_n which is undamped natural frequency 31.30 radian per second or f_n 4.98 hertz, it matches to our solution. You can see damping ratio also matches to our solution. Logarithmic decrement also matches only Z_4 very small difference, it depends up to which decimal place we have approximated our result depending upon that this much difference only we can see. Alright!