

Advanced Composites
Prof. Nachiketa Tiwari
Department of Mechanical Engineering
Indian Institute of Technology, Kanpur

Lecture – 43
Thermal Effects in Composite Laminates (Part-3)

Hello, welcome to Advanced Composites. Today is the start of the 8th week of this course. And over this week we will continue our discussion in the context of solutions for these partial differential equations, which were generated to represent equilibrium of present laminated composite plates.

While last week we have focused exclusively on semi infinite plates and their solutions. And at the close of the last week we started touching up on the issue of thermal stresses and thermal strains, because these types of stresses and strains are very important specially in context of laminated composite plates because, they are associated with the manufacturing of these plates.

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$N_{xy}^+ = 0$ $\sigma_{x,eff}^+ = 0 = \sigma_x(y) + \frac{\tau}{2y}$
 If the plate experiences $\Delta T \rightarrow$ DEFORMED SHAPE?

STEP 1: EQUILIBRIUM EQNS: $\frac{\partial}{\partial x} = \frac{d}{dx}$ $\frac{\partial}{\partial y} = 0$

$\frac{dN_x}{dx} = 0 \rightarrow N_x = C_1 = 0$ from $N_x^+ = 0$
 $\frac{dN_{xy}}{dx} = 0 \rightarrow N_{xy} = C_2 = 0$ from $N_{xy}^+ = 0$

$\frac{d^2 M_x}{dx^2} = 0 \rightarrow M_x = C_3 x + C_4$ from $M_x^+ = 0 \rightarrow C_3 = C_4 = 0$
 $M_x = 0$

A-B-D RELATIONS: $A_{11} \quad B_{11} \quad A_{22} \quad B_{22} \quad D_{11} \quad D_{22} \quad B_{12} \quad B_{21} \quad D_{12} \quad D_{21} \quad D_{66} \quad B_{16} \quad B_{61} \quad B_{26} \quad B_{62} \quad D_{16} \quad D_{61} \quad D_{26} \quad D_{62}$

So, in that context we had developed equations of equilibrium and also modified ABD relations for these types of plates which have inbuilt thermal stresses and strains; and then we started to solve a particular problem related to semi infinite plates so that it helps us understand the issues at hand better in context of thermal stresses. So, the problem which we were discussing was this one. And here we have an infinite semi infinitely long

beam. The dimension in the length direction is a , the dimension in the other direction that is it is with b .

And we have assumed that a is extremely large compared to b and the plate is free to move so it is not constrained on either of the edges. So, because it is free to move the boundary conditions associated with such a plate are given here that on both the edges x is equal to plus minus a over to the external value of N_x that is N_x plus or N_x minus is 0 also N_{xy} N_x and N_x plus is 0 and also bits because it is not seen any external moments or shear forces. So, M_x is 0 and also Q_x effect is 0.

So, for these boundary conditions there will be proceeded to solve the governing differential equations. From the first differential equation which represents equilibrium of forces in the x direction; we got N_x is equal to C_1 equals 0; from the second governing equation we got N_{xy} equals C_2 equals 0. So, these are the two governing integrations of the first two other governing equations.

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Handwritten mathematical derivation on a whiteboard showing the equilibrium equations for a plate. The equations are:

$$N_x = A_{11} \frac{du}{dx} - B_{11} \frac{d^2w}{dx^2} - N_x^T \Delta T = C_1 = 0 \quad (1)$$

$$N_y = A_{12} \frac{du}{dx} - B_{12} \frac{d^2w}{dx^2} - N_y^T \Delta T = 0 \quad (2)$$

$$N_{xy} = A_{66} \frac{dy}{dx} - N_{xy}^T \Delta T = C_2 = 0 \quad (3)$$

$$M_x = B_{11} \frac{dw}{dx} - D_{11} \frac{d^3w}{dx^3} - M_x^T \Delta T = 0 \quad (4)$$

A-B-D RELATIONS: $A_{16} B_{16} A_{26} A_{26} D_{16} D_{26} B_{12} B_{20} = 0$ CROSS-PLY.

$N_{xy} = 0$ $N_{xy}^T = 0$

$C_3 \rightarrow$ Right Body TRANSLATION

And from the third governing differential equation which represents equilibrium of forces in z direction as well as equilibrium of moments along x and y axis. We got $d^2 M_x$ over dx^2 equals 0, and then we integrate that we got M_x equals $C_3 x$ plus C_4 and because M_x is equal to 0 on both the edges of the plate at x is equal to plus minus a , plus minus a over 2. The values of integration constants C_3 and C_4 , they turn out to be 0. So, M_x is equal to 0. So, in overall sense N_x equals 0, N_{xy} equals 0 and M_x

equals 0; and that makes sense also because the plate which we are discussing is not seeing any external forces and it is free to move. So, we should not expect any internal forces and moments in the plate.

So, once these equilibrium equations were integrated, then we proceeded to linking these values of N_x , N_{xy} , M_x , M_{xy} and so on and so forth with mid plane strains and curvatures. And through mid planes strains and mid planes curvatures to mid plane displacements u , v and w naught. So, what we had done in the last class was that we integrated the we stated that N_x equals $A_{11} \frac{du}{dx}$ minus $B_{11} \frac{d^2w}{dx^2}$ minus $N_x^T \Delta T$ equals C_1 . Similarly, we also developed an expression for an N_y and N_{xy} .

And from the second equation this one is said that I am sorry from third equations for N_{xy} , we said we came up with an expression that V naught equals C_3 over A_{66} and that equals to be 0. And then we were working on the fourth equation for M_x , so that states M_x equals $B_{11} \frac{du}{dx}$ minus $D_{11} \frac{d^2w}{dx^2}$ minus $M_x^T \Delta T$. And now what will do is we will write down the other two relations for M_{xy} and M_y .

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The image shows a whiteboard with handwritten equilibrium equations for a plate. The equations are numbered 1 through 6. Equation 1 is $N_x = A_{11} \frac{du}{dx} - B_{11} \frac{d^2w}{dx^2} - N_x^T \Delta T = C_1 = 0$. Equation 2 is $N_y = A_{22} \frac{dv}{dy} - B_{22} \frac{d^2w}{dy^2} - N_y^T \Delta T = C_2 = 0$. Equation 3 is $N_{xy} = A_{66} \frac{du}{dx} - N_{xy}^T \Delta T = C_3 = 0$, with a note $\therefore C_3 \rightarrow \text{RIGID BODY TRANSLATION}$. Equation 4 is $M_x = B_{11} \frac{du}{dx} - D_{11} \frac{d^2w}{dx^2} - M_x^T \Delta T = 0$. Equation 5 is $M_y = -D_{12} \frac{d^2w}{dx^2} - M_y^T \Delta T$. Equation 6 is $M_{xy} = 0$. There are green checkmarks next to equations 1 and 4, and a pink arrow pointing to equation 2.

So, M_y equals minus $D_{12} \frac{d^2w}{dx^2}$ because why is it that because all the b s associate terms for the relation of M_y , they involve what D_{12} which is 0, D_{16} which is 0, and B_{66} which is 0. So, B terms do not come into the picture and for M_y

if the D terms associated are D_{12} so that is non 0 and so is $d^2 w$ over dx^2 . And then the next term would be D_{22} times curvature in y direction which is 0 because the plate is semi infinitely long in x direction. And the third term will be D_{16} times $k \cdot x \cdot y$, so D_{16} is 0. So, this is the only term which is left. And the and of course, there is a temperature related term, so that is $M_y \hat{T} \cdot \Delta T$. And the sixth equation is M_{xy} equals and when we expanded we find that either in all the terms equals 0, so this is equal to 0. So, these are the 6 equations.

Now, our aim is to find expressions for u naught, v naught and w naught. v naught we have already showed that out that values works out to be exactly 0 throughout the length of the plate. So, for u naught and w naught we consider equations 1. So, when we observe equation 1, it has u naught and w naught and everything else in the equation is known, A_{11} is known, B_{11} is known, $M_x \cdot T$ we can compute because we know the lamination sequence and α_1 and α_2 of the material and ΔT is also known.

So, the first equation involves only u naught and w naught. And the fourth equation is same is true for fourth equation also. So, essentially what we will do is we will integrate these equations with respect to x . So, we will get an expression in u and w dw over dx and then solve for u naught and dw/dx .

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The image shows a whiteboard with handwritten mathematical equations. The first equation is labeled 'From ①' and is
$$v^0(x) = \frac{B_{11}}{A_{11}} \frac{dw}{dx} + \frac{N_x^T \cdot \Delta T \cdot x}{A_{11}} + \frac{C_5}{A_{11}} \quad \text{--- (7)}$$
 The second equation is labeled 'From ④' and is
$$v^0(x) = \left[\frac{B_{11}}{A_{11}} \frac{dw}{dx} + \frac{N_x^T \cdot \Delta T \cdot x}{A_{11}} + \frac{C_5}{A_{11}} \right] \cdot \frac{1}{B_{11}} \quad \text{--- (8)}$$
 Below these equations, there is a note: 'Two eqn in $v^0(x)$ and dw/dx .'

So, we get from 1 we get u naught x is equal to B_{11} over A_{11} dw naught over dx plus $N_x \cdot T \hat{T} \cdot \Delta T \cdot x$ because that I have integrated it once. And then I get an

integrated integration constant, so that is C_5 over A_1 . The other thing is so from likewise from 4, we get another expression for u_{naught} and that gives us $D_1 \frac{dw_{naught}}{dx}$ oh so they should be here also A_1 . And from the fourth equation, what we get is D_1 times $\frac{dw_{naught}}{dx}$ plus this is the equation we are considering. So, $M \times T \Delta T$ $M \times T$ times ΔT times x plus another integration constant C_7 and this entire thing gets multiplied by $\frac{1}{B_1}$. So, this is equation 7 and this is equation 8.

Now, consider the case that if ΔT is 0, if ΔT is 0 then the plate should not exhibit any u_{naught} . If ΔT , if temperature is 0, the only reason plates configuration is going to change it is free, it has not any external forces, so the only reason plate is going to change it is configuration and exhibit u v and w is because of temperature difference. So, in case ΔT is 0 then u_{naught} should be 0 also the plate should not exhibit any curvature, so $\frac{dw_{naught}}{dx}$ should be 0.

So, essentially what am trying to say is that this thing will be 0, this term will be 0 and because ΔT is 0 this should be 0. So, the own an which implies that C_5 again represents rigid body motion and we can set it as 0. So, we said this as 0. Similarly, we look at equation four and we see that and we again do the same thought experiment that when ΔT is 0, u_{naught} should be 0, $\frac{dw_{naught}}{dx}$ should be 0 so and the only nonzero term left is C_7 . So, for this equation to be satisfied this also has to be set to 0 because C_7 and C_5 represent rigid body translation of the plate.

So, once we said these to be 0 what we see is that these modified equations after they are said to be 0 or equations in u_{naught} and $\frac{dw_{naught}}{dx}$. So, we have two equations in u_{naught} x and $\frac{dw_{naught}}{dx}$. These are simultaneously equation simultaneous equations. So, we can solve for u_{naught} and $\frac{dw_{naught}}{dx}$.

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From (4)

$$V^0(x) = \left[D_{11} \frac{dw^0}{dx} + \hat{M}_x^T \cdot \Delta T \cdot x + \frac{1}{2} x^2 \frac{1}{B_{11}} \right] - (8)$$

Two eqn in $V^0(x)$ and dw^0/dx .
Solve for $V^0(x)$ and dw^0/dx

So, we solve for u naught x and dw naught over dx . So, when we solve for this is what we get.

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$$V^0(x) = \left[\frac{B_{11}}{A_{11} \cdot D_{11}} \left\{ \frac{D_{11} \hat{N}_x^T}{B_{11}} - \hat{M}_x^T \right\} \right] \Delta T \cdot x$$

$$\frac{dw^0}{dx} = \left[\frac{B_{11}}{D_{11} \cdot A_{11}} \left\{ \hat{N}_x^T - \hat{M}_x^T \cdot \frac{A_{11}}{D_{11}} \right\} \right] \cdot \Delta T \cdot x$$

$$W^0(x) = \left[\frac{B_{11}}{D_{11} \cdot A_{11}} \left\{ \hat{N}_x^T - \hat{M}_x^T \cdot \frac{A_{11}}{D_{11}} \right\} \right] \cdot \Delta T \cdot \frac{x^2}{2} + C_8^0$$

$V^0(x) = 0$

$\begin{bmatrix} 0 & | & q_0 \end{bmatrix} \rightarrow \begin{bmatrix} 0 & | & q_0 \end{bmatrix}_s$

$\begin{bmatrix} \hat{N}_x^T \\ \hat{M}_x^T \end{bmatrix}$

U naught x equals $B_{11} / (A_{11} \cdot D_{11})$ times $D_{11} \cdot N_x^T / A_{11} - M_x^T$ hat this entire thing multiplied by $\Delta T \cdot x$. And dw over dx works out to be $1 / (D_{11} \cdot A_{11})$ times so that is times ΔT times x , so that is the slope of the plate at any given point. So, I integrate so this is so this is one important equation. And this is the equation the next equation is for slope. And if I integrate this equation for slope that is

dw/dx , I get w as a function of x is equal to this entire expression times ΔT times $x^2/2$ plus another integration constant C_8 .

So, once again what is how do we find out the value of C_8 , C_8 , we again think of a scenario that when ΔT is 0, then the plate should not bend at all and that can happen if C_8 is 0. So, this is 0. And C_8 represents rigid body motion of the plate; it does not represent any deformation of the body. So, this is the expression for u , then we also have an expression for v and then so v is equal to 0. So, these are the relations.

Now, we will discuss these relations for a couple of minutes, and then move onto the next topic. So, consider these and think about a scenario that when ΔT is 0, when ΔT is 0 what does the equation tell us when ΔT is 0 then all these terms they may not be 0. But if ΔT is 0 then all these terms are not 0, but they get multiplied by ΔT , so u is 0. So, when there is no temperature change then there is no u and which makes sense. Similarly the same is true for the second case also, because the entire solution is a multiple of ΔT , so that is one thing to understand.

The second thing I would like to state is that suppose the plate was symmetric suppose the plate was in this case we said that the plate is un-symmetric. So, it is 0, 90. And because of this type of plate because of this type of plate 0 90 the plate will tend to bend when I heat it. It is like a bimetallic plate the side glue and piece of aluminum steel together and when I heat it, it will try to bend it is bimetallic because different materials have different coefficients of thermal expansion in different directions. So, this kind of plate will bend.

Suppose the plate was symmetric let us say it is 0 90 symmetric, and then we heat it, then we would not expect the plate to bend, but ΔT will be non-zero, but then what will make the plate not bend. The reason in that case will be that when we compute the values of N_x and M_x these values they will work out to be 0, when we compute these values. Because what are the definitions of these things, so if we compute these things we will find that these terms will work out to be 0. And once that happens so what was I was saying this that this ΔT as it becomes 0 the plates do not exhibit any deformation.

The other thing is what happens when the plate is symmetric and if ΔT is non-zero, then what would happen. So, in that case we should look at these relations for u and w , so when the plate is symmetric in both the expressions for u and w we have in the new replicator both the relation for u , the entire displacement function is multiplied by this term B_{11} and when the plate is symmetric then B_{11} will be 0, so u will be 0. Similarly, when the plate is symmetric B_{11} is 0. So, when B_{11} here is also an expression for w . So, again w will also be 0.

So, this again needs a physical requirement. And in this way very methodically and logically we can develop expressions for mid plane stresses and mid plane strains and thermal stresses and thermal strains in context of plates which are un symmetric and if they are treated to temperatures. We can also use the same approach to calculate internal stresses for even symmetric systems when they are subjected to temperatures, so that is what I wanted to discuss today. And tomorrow we will move onto our different topic and we will start looking at plates which are not necessarily semi infinite in dimensions. So, we will start approaching plates of those types and then we will see how they can be solved.

Thank you very much and have a great day.