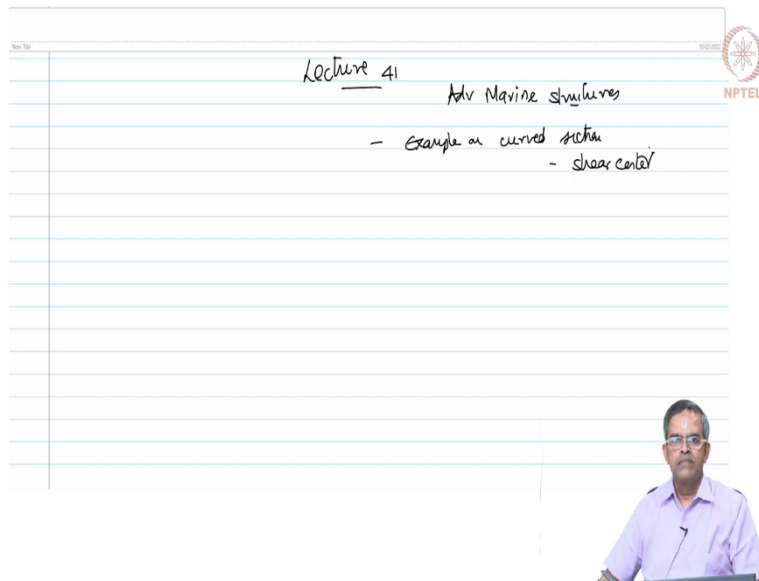


Advanced Design of Steel Structures
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Lecture - 41
Curved Section

(Refer Slide Time: 00:20)



Friends, welcome to the lecture 41 of Advanced Marine Structures. In this lecture, we are going to learn example on Curved Sections to Locate the Shear Center. So, we already studied locating shear center from the first principles simple exercise provided a section has got at least one axis of symmetry. If axis of symmetry does not exist, then the section will be subject to unsymmetric bending. Then, we have got to combine both the problems of unsymmetric bending, locate the u u v v axis, then find the shear center. I will come to that problem slightly later. We will do that.

So, in the previous examples, we discussed about rectilinear sections, like I section, channels, t, etcetera. In this example, it is quite interesting we will take a curved section because we are going to slowly understand the analysis and design of curved beams and crane hooks. So, we need to at least examine the curved section for shear center calculations. Let us do that. We will take I will call this example 1 for this lecture. We will talk about curved sections.

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Example 1 - curved section

Given data: $R, t, 2\alpha, \beta$

shear stress, $\tau = \frac{V}{It} \int y dA$

$\tau = \frac{V}{It} \int_{\beta}^{\theta} R \cos \theta R t d\theta$

Elemental shear stress, $\tau = \frac{VR^2}{I} (\sin \theta - \sin \beta)$

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$e = ?$

$\frac{\pi}{2} - \beta = \alpha$

So, now, let us say, I have an axis of symmetry of the section. We will mark a curved section symmetric about this axis. So, let us say it has got a center line, and that center line is at a distance R , from here and I call this point as O , ok, this point as O . Let us say it subtends an angle 2α symmetric about the center, so this angle is α . Let us believe the section has got uniform thickness t and the curvature start of this section from the axis of rotation is taken as β .

So, now; obviously, being this the axis of symmetry shear center will lie on this axis. Let us mark the shear center somewhere here. If the force and the reaction pass through the shear center the section will not be subjected in distinct moment. We call this distance of the shear center from the axis of curvature as e . So, the question is now I want to find e . The given data for the section are the following, radius of the section, thickness of the section, the included angle and the subtended angle. Let me write it clear; are known thing.

So, let us now cut the section which is subtending an angle $d\theta$ at θ from here. Now, let us say this is my hatched portion. Now, the shear stress τ is given /the classical equation V / It integral $y dA$. Now, my question of integration is going to be, for the element is going to be varying from β to θ . That is my control, is it not? That is my dA . Look at here, this is my dA , right. That is what I am looking at.

So, let me do that which will be equal to V / It , integral β to θ , y will be $R \cos \theta$ and $R t \theta d\theta$, $t d\theta$, $R d\theta t$ will be the area of the hatched portion which is green in color. This is y bar. So, let

us we can do this. So, the elemental shear stress τ is equal to $VR^2 / I, \sin\theta - \sin\beta$, am I right. That is my elemental shear stress. Also, friends, we have an interesting relationship here. From this figure we can say $\pi/2 - \beta$ is α . Can I say that? From this figure, from the geometry this is true.

(Refer Slide Time: 08:16)

The slide contains the following handwritten derivations:

$$\tau = \frac{VR^2}{I} (\sin\theta - \sin\beta)$$

Elemental shear force, $dv = \tau da$

$$dv = \frac{VR^2}{I} (\sin\theta - \sin\beta) R t d\theta$$

$$dv = \frac{VR^3 t}{I} (\sin\theta - \sin\beta) d\theta$$

Moment of this force, about the center, $dM = \frac{VR^3 t}{I} (\sin\theta - \sin\beta) d\theta (R)$

$$dM = \frac{VR^4 t}{I} (\sin\theta - \sin\beta) d\theta$$

Total Moment, $M = 2 \times \int_{\beta}^{\pi/2} dM$

The slide also features the NPTEL logo in the top right corner and a small video inset of a man in a light blue shirt speaking in the bottom right corner.



Having said this, so we said that τ is $VR^2 / I \sin\theta - \sin\beta$. Now, I can find the elemental shear force, which is dv , which will be actually τda . So, now, I can find dv as $VR^2 / I, \sin\theta - \sin\beta$ into $R t d\theta$. So, now, this becomes $VR^3 t / I, \sin\theta - \sin\beta$ of $d\theta$. This is my dv . Now, the moment of this shear force about the center is given by dM . dM will be straight away into R , is it not. The distance is R radius.

So, I can say $VR^3 t / I, \sin\theta - \sin\beta d\theta$ multiplied by R . So, this will become $VR^4 t / I, \sin\theta - \sin\beta$ into $d\theta$. That is my dM . So, once we obtain the elemental moment, we can find the total moment, yeah which will be integral of dM varying from β to $\pi/2$, 2 times of that.

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$$M = 2 \int_{\beta}^{\pi/2} dM = 2 \int_{\beta}^{\pi/2} \frac{VR^4 t}{I} (\sin \theta - \sin \beta) d\theta$$

$$M = \frac{2VR^4 t}{I} \left[-\cos \theta - \theta \sin \beta \right]_{\beta}^{\pi/2}$$

$$M = \frac{2VR^4 t}{I}$$



So, the total moment M is twice of integral β to $\pi/2$ dM which will be twice of integral β to $\pi/2$ $VR^4 t / I, \sin \theta - \sin \beta d\theta$. So, which will give me M as $2 VR^4 t / I$ of $-\cos \theta - \theta \sin \beta$, apply the limits from β to $\pi/2$. So, that will give me M as $2 VR^4 t / I$.



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$$M = 2 \int_{\beta}^{\pi/2} dM = 2 \int_{\beta}^{\pi/2} \frac{VR^4 t}{I} (\sin \theta - \sin \beta) d\theta$$

$$M = \frac{2VR^4 t}{I} \left[-\cos \theta - \theta \sin \beta \right]_{\beta}^{\pi/2}$$

We also know, $\left(\frac{\pi}{2} - \beta = \alpha\right)$

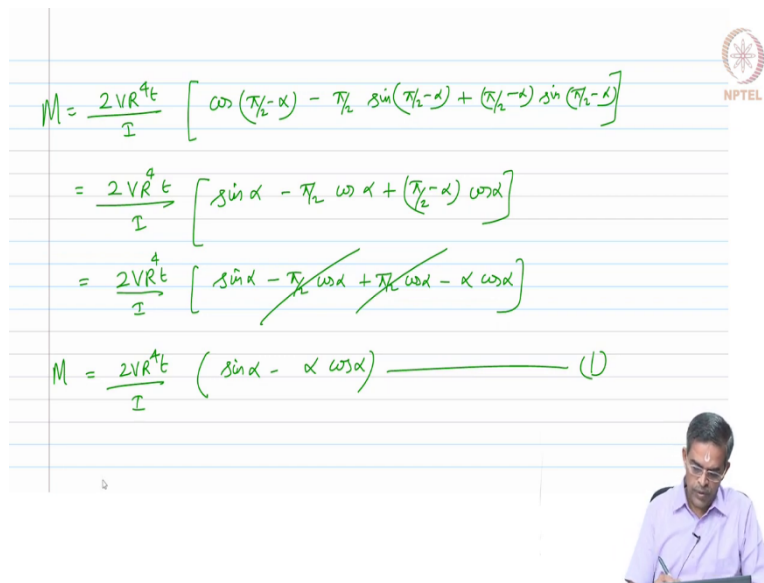
$$M = \frac{2VR^4 t}{I} \left[-\cos\left(\frac{\pi}{2}\right) + \cos \beta - \frac{\pi}{2} \sin \beta + \beta \sin \beta \right]$$

$$= \frac{2VR^4 t}{I} \left[\cos\left(\frac{\pi}{2} - \alpha\right) - \frac{\pi}{2} \sin\left(\frac{\pi}{2} - \alpha\right) + \left(\frac{\pi}{2} - \alpha\right) \sin\left(\frac{\pi}{2} - \alpha\right) \right]$$



Before that we also know $\pi/2 - \beta$ is α . Can you see this here? Let me use that relationship here. So, now, saying that M will now become $2 VR^4 t / I - \cos \pi/2 + \cos \beta - \pi/2 \sin \beta + \beta \sin \beta$. So, we can say this is 0. And we have $\cos \beta \sin \beta$, right. Now, I can simplify this equation as $2 VR^4 t / I$. I am replacing β as $\pi/2 - \alpha$.

So, can I write this as $\cos\pi/2 - \alpha - \pi/2 \sin\pi/2 - \alpha + \pi/2 - \alpha$ of $\sin\pi/2 - \alpha$. I am bringing all in terms of α . Let me copy this equation to the next screen.

(Refer Slide Time: 14:08)



The slide shows a handwritten derivation of the moment M. The steps are as follows:

$$M = \frac{2VR^4t}{I} \left[\cos\left(\frac{\pi}{2} - \alpha\right) - \frac{\pi}{2} \sin\left(\frac{\pi}{2} - \alpha\right) + \left(\frac{\pi}{2} - \alpha\right) \sin\left(\frac{\pi}{2} - \alpha\right) \right]$$

$$= \frac{2VR^4t}{I} \left[\sin\alpha - \frac{\pi}{2} \cos\alpha + \left(\frac{\pi}{2} - \alpha\right) \cos\alpha \right]$$

$$= \frac{2VR^4t}{I} \left[\sin\alpha - \cancel{\frac{\pi}{2} \cos\alpha} + \cancel{\frac{\pi}{2} \cos\alpha} - \alpha \cos\alpha \right]$$

$$M = \frac{2VR^4t}{I} (\sin\alpha - \alpha \cos\alpha) \quad \text{--- (1)}$$

So, M was equal to this which now becomes $2 VR^4 t / I, \sin \alpha - \pi/2 \cos \alpha + \pi/2 - \alpha \cos \alpha$. Which is further simplified as $2 VR^4 t / I, \sin \alpha - \pi/2 \cos \alpha + \pi/2 \cos \alpha - \alpha \cos \alpha$. So, this goes away; which will now land up as $2 VR^4 t / I \sin \alpha - \alpha \cos \alpha$. We call this equation number 1. It is an interesting equation which is a classical theory available in the literature of a curved sections. Now, let us take moment about O. This is the figure we have. Let us copy this figure.

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Take moment about 'O'

$$V e = M$$

$$= \frac{2VR^4 t}{I} (\sin \alpha - \alpha \cos \alpha)$$

$$e = \frac{2R^4 t}{I} (\sin \alpha - \alpha \cos \alpha)$$

I = MoI of the whole section ?



We will take moment about O. So, V into e is equal to that moment. So, which I say is equal to $2 VR^4 t / I \sin \alpha - \alpha \cos \alpha$. That is what the moment we have here.. I have equated this. So, now, I can easily get e as sorry, $2 R^4 t / I \sin \alpha - \alpha \cos \alpha$ that is my e . If we know the value of e , I can easily find the shear center, locate the center. So, look at this equation, in this equation R is a geometric property, t is a section property, α is the angle of this curved section subtended the center to α rather.

But the question is how about I . I is a moment of inertia of the whole section. Is it not? That is what we understand. It is a segmented section; it is not a full curve. So, that we can find I readily; the segmented section. So, the problem is how to find this. Let us do that.

(Refer Slide Time: 18:03)

To find MoI (I) of the segment

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$$I = \int y^2 dA$$

$$= 2 \int_{\beta}^{\pi/2} (R \cos \theta)^2 (R d\theta t)$$

$$= 2 R^3 t \int_{\beta}^{\pi/2} \cos^2 \theta d\theta$$

We know that $\cos 2\theta = 2 \cos^2 \theta - 1$
 $\therefore \cos^2 \theta = \frac{1 + \cos 2\theta}{2}$

Substituting, $I = \frac{2 R^3 t}{2} \int_{\beta}^{\pi/2} (1 + \cos 2\theta) d\theta$

Now, the issue is to find the moment of inertia I of the segment. So, let us draw this figure again. I will take this figure again. We know I is y square dA, and I want to integrate it for the entire section. So, when I integrate I can do this as integration of β to $\pi/2$ and 2 times of this. y we already have with us, it is $R \cos \theta$, am I right. Look at this figure, $R \cos \theta$ will be the y value. That is this value, this is R.

And this value $R \cos \theta$, is it not. So, $R \cos \theta$ the whole square and dA is $R d\theta$ into t, am I right? So, now, let us simplify this, so $2 R^3 t \int_{\beta}^{\pi/2} \cos^2 \theta d\theta$. We know from trigonometry, that $\cos 2\theta$ is $2 \cos^2 \theta - 1$. So, hence $\cos^2 \theta$ is $\frac{1 + \cos 2\theta}{2}$. So, substituting, I is $2 R^3 t / 2 \int_{\beta}^{\pi/2} (1 + \cos 2\theta) d\theta$.

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$$I = R^3 t \int_{\beta}^{\pi/2} (1 + \cos 2\theta) d\theta$$

$$= R^3 t \left[\theta + \frac{\sin 2\theta}{2} \right]_{\beta}^{\pi/2}$$

$$= R^3 t \left[\frac{\pi}{2} - \beta + \frac{\sin(\pi/2)}{2} - \frac{\sin 2\beta}{2} \right]$$

we know that $\pi/2 - \beta = \alpha$ $\sin 2A = 2 \sin A \cos A$

$$I = R^3 t \left[\underbrace{\left(\frac{\pi}{2} - \beta \right)}_{\alpha} + \frac{1}{2} - \frac{1}{2} \sin(2(\pi/2 - \alpha)) \right]$$

So, I is R cube t integral β to $\pi/2$, $1 + \cos 2\theta$ d θ , which is R cube t $\theta + \sin\theta / 2$ β to $\pi/2$, which is R cube t $\pi/2 - \beta + \sin 2\pi/2 / 2, - \sin 2\beta / 2$. We know already $\pi/2 - \beta$ is α , is it not. We know already have it here, see here. Hence, I will now become R cube t, say $\pi/2 - \beta$. So, this becomes α , half of $0 - \text{half of } \sin \text{ twice of } \pi/2 - \alpha$. Can I say that? So, this is closed.

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$$I = R^3 t \left\{ \alpha - \frac{1}{2} (2 \sin(\pi/2 - \alpha) \cos(\pi/2 - \alpha)) \right\}$$

$$= R^3 t \left\{ \alpha - \cos \alpha \sin \alpha \right\} \quad (2)$$

$$C = \frac{2 R^4 t}{I} (\sin \alpha - \alpha \cos \alpha)$$

$$\Rightarrow C = \frac{2 R^4 t}{R^3 t} \left[\frac{(\sin \alpha - \alpha \cos \alpha)}{(\alpha - \sin \alpha \cos \alpha)} \right] = 2R \frac{(\sin \alpha - \alpha \cos \alpha)}{(\alpha - \sin \alpha \cos \alpha)}$$

Notice the shear cube

So, now I can say I is R cube t. See, look at this figure. So, this becomes α - this is half of $\sin 2\theta$, so $2 \sin\theta$ are $R\theta$. So, half of $2 \sin\pi/2 - \alpha, \cos\pi/2 - \alpha$. Can I write like this? This is argument is 2θ , let us say $\sin 2\theta$ is $2 \sin\theta \cos\theta$. That is what I am writing here. So, these two goes away now, which will become R cube t $\alpha - \cos \alpha \sin \alpha$. So, this is equation number 2.

So, now, e is $2 R^4 t / I, \sin \alpha - \alpha \cos \alpha$. That is what we have written here, $2 R^4 t / I \sin \alpha - \alpha \cos \alpha$. So, now, therefore, substituting e will become $2 R^4 t / R^3 t, \sin \alpha - \alpha \cos \alpha / \alpha - \sin \alpha \cos \alpha$. So, which will now become $2 R$ times of $\sin \alpha - \alpha \cos \alpha / \alpha - \sin \alpha \cos \alpha$, that is my e . So, we have all the values. So, geometric parameters, I can find e .

Now, I can locate the shear center for the given geometry. So, as a small example, let us copy this figure, somewhere see this figure.

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Let

$R = 600 \text{ mm}$

$2\alpha = 60^\circ$

find e

$$e = 2R \left(\frac{\sin \alpha - \alpha \cos \alpha}{\alpha - \sin \alpha \cos \alpha} \right)$$

$= x \text{ mm}$

$e = x \text{ mm}$

So, let us say, let R be 600 mm . A or 2α be 60 degrees. Find e . So, e is straight away $2 R$ times of $\sin \alpha - \alpha \cos \alpha / \alpha - \sin \alpha \cos \alpha$. And compute this, mark the value and e will be maybe x millimeters. So, friends, it is very simple that this particular numerical example explains how to compute the shear center for a curved section as you see here.

(Refer Slide Time: 26:54)

Ex 2 locate the shear center

By sym, $V_1 \equiv V_5$
 $V_2 \equiv V_4$

To find V_1

$$V_1 = \int_0^{b_1} \frac{V}{It} (a\bar{y}) da$$

$$a = tz \quad \bar{y} = (R + b_1 - z/2)$$

$$da = t dz$$

We will do one more example. By a slightly a tricky section, let us see how we handle this section. Locate the shear center of the section shown. So, section we have, ok; let us marks on the dimensions of this section. Of course, we can locate there is an axis of symmetry for the section. The section has uniform thickness as t . Let us mark the center line of the section for our calculation purposes.

And we mark this dimension as b_1 and this dimension as b and this radius as R . We will say the shear center is located somewhere here, and this is at an eccentricity e from here. Let us mark the shear flow. We call this arm as V_5 , this as V_4 , this as V_3 , this as V_2 and this as V_1 , this as V_1 . So, from this figure, we can easily say by symmetry, V_1 will be same as V_5 . Further, V_2 will be same as V_4 . Let us cut the section at z from here, that is the thickness of the strip $b dz$. This what we hatch. We want to find the cg of this, ok from here axis of symmetry.

From the figure, you can always write this value as $R + b_1 - z/2$. Can I say that? Can I say that? So, now, I can find V_1 . V_1 will be the force in the vertical arm which will be integrated from 0 to b_1 , V/It , a y bar of da . So, a will be t into z and da will be t into d , and y bar will be $R + b_1 - z/2$. I can substitute this.

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The slide shows a derivation for the velocity V_1 in a curved section of a pipe. The diagram illustrates a pipe with a semi-circular bend of radius R and a straight section of length b . The pipe has an inner radius b_1 and a wall thickness t . A coordinate system is defined with z along the length of the pipe and r as the radial distance from the center of the bend. The velocity profile V_1 is shown as a function of z .

$$V_1 = \int_0^{b_1} \frac{V}{It} \left(R + b_1 - \frac{z}{2} \right) t dz$$

$$= \frac{Vt}{I} \int_0^{b_1} \left(R + b_1 - \frac{z}{2} \right) z dz$$

$$= \frac{Vt}{I} \left(\frac{Rb_1^2}{2} + \frac{b_1 b_1^2}{2} - \frac{b_1^3}{6} \right)$$

$$V_1 = \frac{Vt}{I} \left(\frac{Rb_1^2}{2} + \frac{b_1^3}{3} \right) \quad (1)$$

So, V_1 is going to be integral 0 to b_1 , V/It , $t z$, $R + b_1 - z/2$ of $t dz$. Let me rub this. So, we can straight away say it is Vt/I now integral 0 to b_1 , $R + b_1 - z/2$ of $z dz$, which can be Vt/I , $Rb_1^2/2 + b_1 b_1^2/2 - b_1^3/6$. After substituting the limits, which becomes Vt/I of $Rb_1^2/2 + b_1^3/3$. This is my V_1 , is equation 1. Ok, friends.

So, we will continue this problem in the next lecture. But I want you to do a small homework on this. I want you to find V_2 . And now we have a curved section, we can find V_3 also. So, try to do this exercise, then we will recollect it how we are able to do it. I leave a small gap here intentionally, so that I want you to practice this. And try to do for V_2 then V_3 and so on.

Thank you very much. And have a good day. Bye.