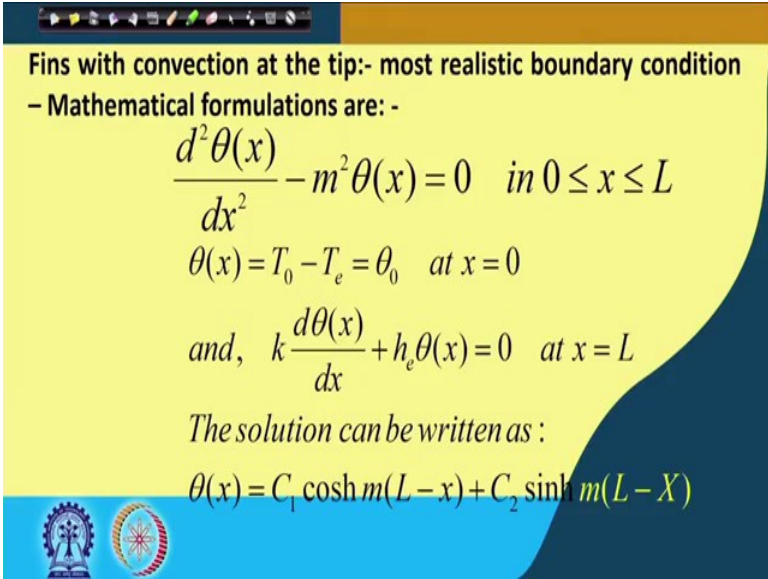


Thermal Operations in Food Process Engineering: Theory and Applications
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Lecture - 19
Finned Surfaces (Contd.)

So, we are discussing about the finned heat transfer analysis. We have done for fin with long fin or we have also done for fin with negligible heat loss at the fin tip. And, these were the two situations which we said and we were left with the third that is fin with the convective boundary, 'right'; obviously, that fin with the convective boundary will not be that easy. So, we have come to the 19th class lecture number 19 we are handling Finned Surface and this is being continued, 'right'.

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Fins with convection at the tip:- most realistic boundary condition
- Mathematical formulations are: -

$$\frac{d^2\theta(x)}{dx^2} - m^2\theta(x) = 0 \quad \text{in } 0 \leq x \leq L$$
$$\theta(x) = T_0 - T_e = \theta_0 \quad \text{at } x = 0$$

and, $k \frac{d\theta(x)}{dx} + h_e\theta(x) = 0 \quad \text{at } x = L$

The solution can be written as :

$$\theta(x) = C_1 \cosh m(L-x) + C_2 \sinh m(L-x)$$

So, we are now going to that fin with convective boundary, 'right'. So, fins with convection at the tip. So, this is the most realistic boundary condition, why we are saying this is the most realistic boundary condition? Because, that most of the cases as we said that if this is that and if this is the fin tip, 'right', fin.

So, then when we are saying in one case we are saying it is a long fin and in other case we are saying that fin tip is attaining the boundary temperature. These two are more hypothetical than that the fin is under some convective boundary condition. So, that is why this is being said that this situation is more realistic than the other two.

So, we also do solve this one and the mathematical formulation for this can be written, the generalized equation is $\frac{d^2\theta(x)}{dx^2} - m^2\theta(x) = 0$ in $0 \leq x \leq L$ and the boundaries.

The first boundary is

$$\theta(x) = T_0 - T_e = \theta_0 \quad \text{at } x = 0$$

This we said conventionally this is assumed and which we are taken as one boundary. Second boundary that varied with long fin that varied with fin with negligible heat loss at the fin tip. So, this is the third that is the fin with convective boundary condition. So, that boundary condition we can write that conduction must be equal to convection, 'right' or some of the convection and conduction equal to 0 in the fin.

So, here, we write that

$$k \frac{d\theta(x)}{dx} + h_e \theta(x) = 0 \quad \text{at } x = L$$

. And, the solution of heat

transfer, we can write in the form as shown above.

So, this is a solution and we have to find out the values of C_1 and C_2 with the help of the boundary conditions. A first boundary was theta x is equal to $T_0 - T_e$ at theta x is equal to 0 which was equal to theta 0. And, the second solution was theta x is equal to cos; second boundary was $k \frac{d\theta(x)}{dx} + h_e \theta(x) = 0$ at x is equal to L, 'right'.

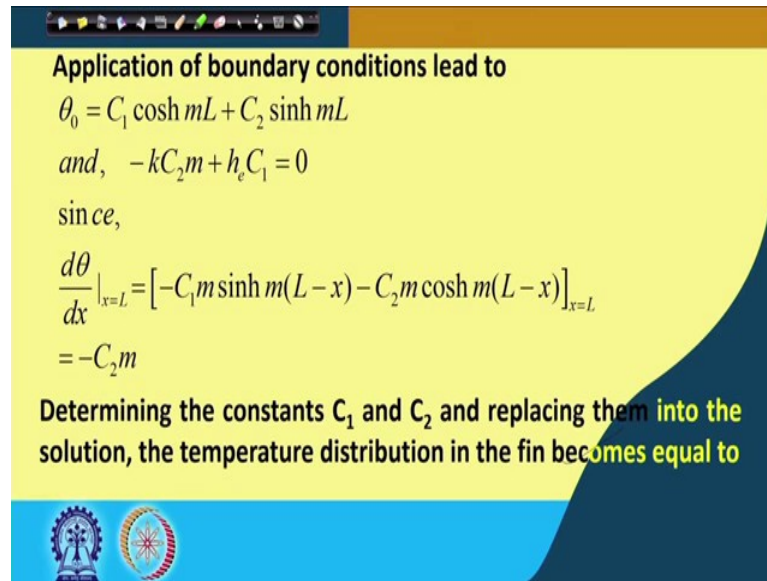
So, if we look at the solution of it the solution came to be equal to that theta x is equal to

We take both this for long with

$$\theta(x) = C_1 \cosh m(L - x) + C_2 \sinh m(L - x)$$

negligible heat loss at the fin tip and this convective boundary both the cases the solution in the form of hyperbolic, 'right'.

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Application of boundary conditions lead to

$$\theta_0 = C_1 \cosh mL + C_2 \sinh mL$$

and, $-kC_2m + h_e C_1 = 0$

since,

$$\left. \frac{d\theta}{dx} \right|_{x=L} = [-C_1 m \sinh m(L-x) - C_2 m \cosh m(L-x)]_{x=L}$$
$$= -C_2 m$$

Determining the constants C_1 and C_2 and replacing them into the solution, the temperature distribution in the fin becomes equal to

So, if we look at that application of boundary conditions to this lead to

$$\theta_0 = C_1 \cosh mL + C_2 \sinh mL$$

and, $-kC_2m + h_e C_1 = 0$

since,

$$\left. \frac{d\theta}{dx} \right|_{x=L} = [-C_1 m \sinh m(L-x) - C_2 m \cosh m(L-x)]_{x=L}$$

$$= -C_2 m$$

So, from these two equations if we look at the solution we can find out the values of C_1 and C_2 'right'. So, determining the constants C_1 and C_2 and replacing them into the solution the temperature distribution in the fin that becomes equal to; so, let us look into that what it is coming in.

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$$\frac{\theta(x)}{\theta_0} = \frac{T(x) - T_e}{T_0 - T_e}$$

$$= \frac{\cosh m(L-x) + (h_e/mk) \sinh m(L-x)}{\cosh mL + (h_e/mk) \sinh mL}$$

and the heat flow rate through the fin is

$$Q = \theta_0 \sqrt{PhkA} \left[\frac{\sinh mL + (h_e/mk) \cosh mL}{\cosh mL + (h_e/mk) \sinh mL} \right]$$

If $h_e = 0$ which corresponds to
no heat loss from the tip leads to

$$\frac{\theta(x)}{\theta_0} = \frac{T(x) - T_e}{T_0 - T_e}$$

$$= \frac{\cosh m(L-x) + (h_e/mk) \sinh m(L-x)}{\cosh mL + (h_e/mk) \sinh mL}$$

and the heat flow rate through the fin is

$$Q = \theta_0 \sqrt{PhkA} \left[\frac{\sinh mL + (h_e/mk) \cosh mL}{\cosh mL + (h_e/mk) \sinh mL} \right]$$

Now, the limiting condition is appearing, 'right', what is that limiting condition which is coming in? So, what is coming?

If h_e , which corresponds to the no heat loss, if h is equal to 0, 'right', that is heat transfer coefficient at the fin surface fin tip, 'right', that is, if h_e is becoming 0; that means, that, there is no heat loss from the fin tip, 'right'. This leads to the solution obtained earlier h_e is 0 meaning that this factor is becoming 0, meaning this whole thing is becoming 0, meaning this whole thing is again becoming 0, meaning it is becoming sin hyperbolic mL over cos hyperbolic mL ; that means, it is becoming tan hyperbolic mL , 'right'.

And, this tan hyperbolic when it is becoming; that tan hyperbolic mL that with the value of L this is becoming close to 1. So, Q is becoming θ_0 under root $PhkA$, 'right'. So, if it is θ_0 under root $PhkA$ then we can say that the whole thing is now equivalent to that fin with negligible heat loss, if h_e that is environmentally transfer coefficient is 0,

'right'. So, that is becoming equivalent to that heat loss with that heat loss with the negligible heat loss at the fin tip, 'right'.

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A one end of a steel rod of diameter $D = 2.5 \text{ cm}$, $L = 25 \text{ cm}$, of thermal conductivity $k = 60 \text{ W / m } ^\circ\text{C}$ is maintained at a temperature of $100 \text{ }^\circ\text{C}$ and the steel rod is exposed to an ambient air at $T_e = 25 \text{ }^\circ\text{C}$ having a heat transfer coefficient $h = 50 \text{ W / m}^2 \text{ }^\circ\text{C}$, calculate the heat loss from the rod.

Solution:- in this case the condition at the other end of the fin is not specified. Let us see the length to diameter ratio which is equal to 10 indicating that we can assume it to be a long fin.

$$m^2 = \frac{hP}{Ak} = \frac{h\pi D}{\left(\frac{\pi D^2}{4}\right)k} = \frac{4h}{kD} = \frac{4 \times 50}{60 \times 0.025} = 133.33$$

$\therefore m = 11.54$ and, $mL = 11.54 \times 0.25 = 2.88$

So, we can say now that one end of a steel with a problem, 'right', we said that we will try to do some problem if possible and if we do some problem; obviously, will be more clear with the things 'right'. So, let us take that a one end of a steel rod of diameter D is equal to 2.5 centimeter, $L = 25$ centimeter, & thermal conductivity k is equal to $60 \text{ W / (m } ^\circ\text{C)}$ is maintained at a temperature of $100 \text{ }^\circ\text{C}$. And, the rod is exposed to an environmental air at T_e is equal to $25 \text{ }^\circ\text{C}$ having a heat transfer coefficient h is equal to $50 \text{ W / (m}^2\text{ }^\circ\text{C)}$.

Now, calculate the heat loss from the rod, 'right'. So, let us repeat a one end of steel rod of diameter 2.5 centimeter, $L = 25$ centimeter, & thermal conductivity k $60 \text{ W / (m}^\circ\text{C)}$ is maintained at a temperature of 100°C . and the steel rod is exposed to an ambient air at T_e is equal to $25 \text{ }^\circ\text{C}$ having a heat transfer coefficient h equal to $50 \text{ W/m}^2\text{ }^\circ\text{C}$; then calculate the heat loss from the rod, 'right'.

So, here we have not been said whether the end of the fin is what under what condition whether it is a long fin or it is a fin with negligible heat loss at the tip nothing has been said, 'right'. So, let us assume that the length of the diameter ratio and which is equal to almost 10 time which is 10 rather indicating that we can assume it to be a long fin, 'right'

because, we are given diameter is 2.5, length is 25. So, L by D that is becoming equal to 10; that means, length is 10 times more than the diameter.

So, if that be true we can assume that it to be a long fin and if we assume it to be a long fin then first thing which we need to know that let us find out what is the value of m, 'right'. So,

$$m^2 = \frac{hP}{Ak} = \frac{h\pi D}{\left(\frac{\pi}{4}D^2\right)k} = \frac{4h}{kD} = \frac{4 \times 50}{60 \times 0.025} = 133.33$$

$$\therefore m = 11.54 \quad \text{and,} \quad mL = 11.54 \times 0.25 = 2.88$$

Now, you remember, we said earlier that tan mL, if mL value is 1 it was somewhere 0.7 if it was 2 if it was 0.96 and if it was 3, it was almost 0.99, 'right'. So, this value has come 2.88; now if that be true then let us find out what is the value of the solution.

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$$\therefore Q = \theta_0 \sqrt{Pakh} = \theta_0 \sqrt{(\pi D) \left(\frac{\pi}{4} D^2\right) kh}$$

$$= (100 - 25) \frac{\pi}{2} \sqrt{(0.025)^3 \times 60 \times 50} = 25.5 \text{ W}$$

If we had used the equation for negligible heat loss, the difference in the result would have been a factor of $\tanh mL = \tanh 2.88 = 0.9937$ which is negligible.

So, Q is equal to then

$$\therefore Q = \theta_0 \sqrt{Pakh} = \theta_0 \sqrt{(\pi D) \left(\frac{\pi}{4} D^2\right) kh}$$

$$= (100 - 25) \frac{\pi}{2} \sqrt{(0.025)^3 \times 60 \times 50} = 25.5 \text{ W}$$

So, if we had used the equation for negligible heat loss the difference in the result would have been a factor of tan hyperbolic mL and that is equal to $mL = \tanh 2.88 = 0.9937$. which is negligible because, this value is 25.5 and this 25.5 into 0.9937 almost would

have been close to 25, 'right'. So that means, in this case first we can assume since, nothing was given we had assumed this is to be a long fin. And, with the solution of the long fin we have found out the value of Q, and the value of Q came out to be 25.5 Watts, 'right'. Now, let us look into the other thing, what could be the other thing? That fin with negligible heat lost at the fin tip.

If we would have taken the difference would have been that $\tanh mL$ 'right'. So, $\tanh mL$, that $\tanh mL$ the value of that when the value of mL became equal to 2.88 means almost close to 3 the $\tanh mL$ almost came to be close to 1, because 0.99 something. So that means, the loss I mean that the difference would have been very negligible, 'right'. So, this is one case where the fin condition was not specified, 'right', but we assumed one, that is we assumed first long fin and then we showed that, if we would have assumed the other one that is fin with negligible heat loss at the fin tip.

Then also the difference between that solution that is the flow of heat that is Q would be negligible, 'right', because that was coming to the factor of 0.99 something, 'right'. So, 0.99 something and the close one is almost same. So, we did not do anything wrong by assuming the first one and we proved then other one also would have lead to the same result or almost very close, 'right'.

So now, let us look into some other situation, other problem that yeah, let us look into other problem which perhaps I missed it here. And, perhaps we will be doing that what is the situation, when we are an linked with different mL or rather we are different situation of the fin, 'right'.

So, that in some class maybe in the next or next to next classes we will be doing with that what are the different types of fins, 'right'; some idea we should have what are the different types of fins are there. So, that we will do perhaps and different types of fin meaning that different types of fin meaning that as we said that, if you assume that this hand is a tube, 'right' and normally the people who make this fin. So, they say that this is an watch fine, but if you just remember, you just imagine that the ladies who are putting what we call fin, 'right', what do you are calling that they are putting some bangles and other things, 'right'.

So, they are pushing it and this fitting is a shrink fitting, this fitting is a shrink fitting and regarding shrink fitting, what it how can it be done? That by some means this say this is

the fin, 'right', this is the fin, 'right' and this diameter has to be actually a little closer or smaller than this one, 'right' then by some means this pipe is made contracted. So, that it becomes shorter in diameter at that situation where, you can put this fin, 'right' like a fin which this one has become shorter. And, you are putting this fin over it, 'right' and it then when it becomes normal then becomes expanded then it fits absolutely perfectly, 'right'.

Now, you remember we said the thermal contact resistance, 'right', if the fin is not properly fixing on it then there will be thermal contact resistance and that will lead to the loss or temperature drop much more. So, which is not desirable that is why this is called shrink fitting; in many cases a round type of a fins are there then they are fitted like that, 'right'. There are that is, 'right' sometime we will also show that what are the different types of fins available and that we will show some day, maybe next class or next to next class depending on how we are doing, 'right'.

So, let us then since our time is over, let us now make it to be a thank you and thank you for your attention. And, but again and again I am saying that do some problem solutions unless you do the solutions of the problems you will not come across, you will not be able to do the solutions or understanding of the situation, 'right'.

Thank you very much.